

DISTILLATION

Separation of two or more components of a mixture by distillation when the materials concerned have a definite vapour pressure, has been one of the most widely used processes, separations which are easy can be accomplished by simple distillation while more difficult separations are made by fractionation which is a process of multiple distillation for effecting the separation of two or more volatile components. Distillation is usually a batch process and may be carried out in a pipe still or a cylindrical vessel. Multiple distillation is carried out in a fractionating column, when continuous interchange between liquid from a condenser and vapour from a still or reboiler leads to the concentration of the lower boiling constituents of a mixture at the top of column. The aim of the column design is to bring about intimate contact between vapour and liquid, while at the same time keeping both streams flowing even in opposite direction.

STRESSES IN VESSELS

defined as a tall vertical vessel, attached to a ship, supporting structures or machinery. Interference caused by any vessel supporting structure in a ship is as a shell. Stresses in the shell of such a vessel may be assessed as follows.

- (a) Circumferential and axial stresses due to internal pressure or vacuum inside the vessel.
- (b) Compressive stresses resulting from dead load such as the weight of the vessel and its contents, the weight of insulation, and attached equipment.
- (c) Stresses resulting from bending moment caused by wind loads acting on the vessel and its attachments.
- (d) Stresses due to eccentricity as a result of irregular load distribution.
- (e) Stresses due to seismic (earthquake) forces.

The above stresses may be combined to establish the final controlling stress. The stresses due to pressure act both in the circumferential and axial direction of the shell.

It is possible to calculate the stresses on the basis of a fixed energy density. However, the pressure in the tanked portion is not the same over the entire height of the shell and the resultant stresses in the shell are, therefore, likely to increase towards the bottom of the shell.

In analysing the stresses at various heights of the shell, calculations are made beginning at the top of the column. The selection and thickness of the top head is based on the internal pressure or vacuum. The minimum shell thickness in the upper portion of the column is usually determined by the circumferential stress resulting from internal pressure or vacuum. The shell plate thickness at the top of the column can be specified by taking into consideration the corrosion allowance. The thickness of the lower portions of the column is determined by taking into consideration all the other stresses.

DETERMINATION OF SHELL THICKNESS (at different heights)

It is also necessary to determine the maximum
stress at various heights of the shell. The
stress at the top of the column is the maximum
stress. The stress at the bottom of the column is
the minimum stress. The stress at the top of the
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The minimum shell thickness in the upper
portion of the column is usually determined
by the circumferential stress resulting from
internal pressure or vacuum. The shell plate
thickness at the top of the column can be
specified by taking into consideration
the corrosion allowance. The thickness
required for any of the column is
determined by taking into consideration
all the other stresses.

DETERMINATION OF SHELL THICKNESS (at different heights)

The usual criterion of shell thickness
is maximum tensile stress which should
not exceed the allowable tensile stress.

which should not exceed the allowable tensile stress in the material of the shell. The maximum thickness is at the top of the shell and is determined only on the basis of circumferential stress

$$t_s = \frac{p D_i}{2 J - p} + c$$

$t_s \rightarrow$ shell thickness

$p \rightarrow$ design pressure

$D_i \rightarrow$ internal diameter of shell

$J \rightarrow$ allowable stress

$J \rightarrow$ joint efficiency

$c \rightarrow$ corrosion allowance

This thickness may be satisfactory up to a certain distance from the top of the shell. In case of vacuum or low pressure the thickness t_s will be much less. A minimum value of thickness may be assumed in such cases

~~REQUIREMENT OF SHELL THICKNESS~~

In order to evaluate the distance X it is necessary to determine the combined maximum stress. The diameter stress at the distance X in the

the shell.

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actual stress in the column is not uniform
 due to the weight of the column above it.

$$P_{ap} = \frac{P D}{4(t - c)}$$

where P is the same parameter throughout
 due to column height
STRESS DUE TO DEAD LOADS

④ COMPRESSIVE STRESS due to weight of shell up to distance 'x'

$$f_{ds} = \frac{\text{weight of the shell}}{\text{cross-section of shell}}$$

$$= \frac{\pi/4 (D_o^2 - D_i^2) \rho_s(x)}{\pi/4 (D_o^2 - D_i^2)}$$

where
 D_o and D_i are Internal and external diameter of shell

$\rho_s \Rightarrow$ density of shell material

$D_m =$ mean diameter of the shell

\Rightarrow weight of the shell insulation per unit height

$$\pi D_m (t_s - c)$$

When a shell is subjected to internal pressure, the maximum stress is at the inner surface of the shell. This maximum stress is assumed to act only on the inner surface of the shell.

$$t_s = \frac{p \cdot d_i}{2f_s - p} + c$$

t_s → shell thickness

p → internal pressure

d_i → internal diameter of shell

f_s → allowable stress

c → joint efficiency

c → corrosion allowance

This thickness may be satisfactory up to a certain distance from the top of the shell. In case of vacuum or low pressure, the thickness t_s will be much less. A minimum value of thickness may be assumed in such cases.

~~REQUIREMENT OF SHELL THICKNESS~~

In order to evaluate the distance x it is necessary to determine the combined maximum stress. The stress at the distance x in the

shell.

axial stresses are given by the following
AXIAL STRESS DUE TO PRESSURE

$$P_{ax} = \frac{P D}{4(t_s - c)}$$

This is the same formula for stress
 due to column height
STRESS DUE TO DEAD LOADS

(a) Compressive stress due to weight of shell
 up to distance 'x'

$$f_{ds} = \frac{\text{Weight of the shell}}{\text{cross-section of shell}}$$

$$= \frac{\pi/4 (D_o^2 - D_i^2) \rho_s(x)}{\pi/4 (D_o^2 - D_i^2)}$$

where

D_o and D_i are Internal and external diameter
 of shell

$\rho_s \Rightarrow$ density of shell material

$D_m \approx$ mean diameter of the shell

\Rightarrow weight of the shell insulation
 per unit height 'x'

$$T D_m (t_s - c)$$

(b) Compressive stress due to liquid in column up to a height 'x'

$$f_d(l) = \frac{\rho_l \times \text{weight of liquid in column up to height } x}{\pi D_m (t_0 - c)}$$

where ρ_l = weight of liquid in column per unit height (x)

D_m , t_0 and c are the diameter, thickness and density of the shell respectively
 D_m is mean diameter of shell
 $D_m \approx D_o$ for large diameter columns

(c) Compressive stress due to liquid in column up to a height 'x'

$$f_d(l) = \frac{\rho_l \times \text{weight of liquid in column up to height } x}{\pi D_m (t_0 - c)}$$

(d) Compressive stress due to attachment such as internals top head, platforms and ladders up to a height of 'x'

$$f_d(att) = \frac{\rho_a \times \text{weight of attachment per unit height } x}{\pi D_m (t_0 - c)}$$

Total compressive dead weight stress

$$f_{dx} = P_{ds} + P_{dcm} + P_{dcr} + P_{dca}$$
 Stresses due to wind load at a distance 'x'

$$f_{wx} = \frac{M_w}{Z}$$

where Z - modulus of section for the cross section of shell

$$= \frac{\pi}{4} D_o^2 (t_s - c)$$

M_w is the bending moment due to wind load at a distance x

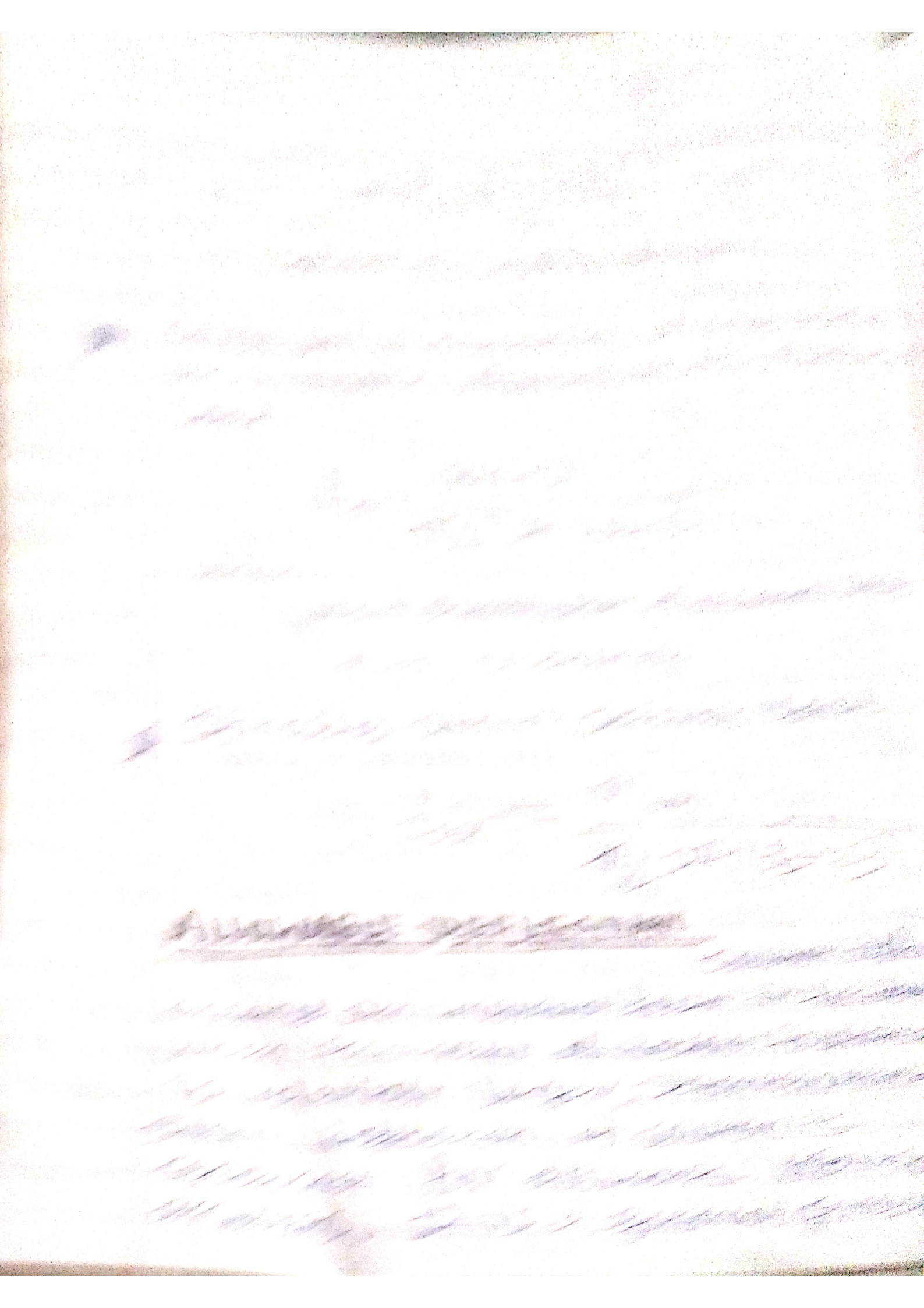
The column is considered as a uniformly loaded cantilever beam

The stress will be compressive on the down wind side and velocity, for

calculations ~~at 125 kg per square meter~~ ~~on a flat surface~~ at 125 kg per square meter on a flat surface

may taken as a satisfactory. For round vessel a shape factor of 0.7 may be used. The projected area of either un-insulated or insulated tower may be taken for operation of wind load

$$M_w = \frac{\text{wind load} \times \text{distance}}{2}$$



To allow a maximum deflection in mm of
 (5 x overall height of column in meter)
 for vertical column, A vibration
 analysis shall be made on column with
 30 meters overall height and above and
 height to diameter ratio ranging from 18 to
 30. The total deflection at the top of
 the column is calculated as an
 algebraic sum by following equation

$$\delta = \sum_1^n \left\{ \left[\frac{WL^3}{8EI} + \frac{(\Sigma W)L^3}{3EI} + \frac{(\Sigma m)L^3}{2EI} \right] + \frac{L}{\sum_1^n} \left[\frac{WL^2}{6EI} + \frac{(\Sigma W)L^2}{2EI} + \frac{(\Sigma m)L}{EI} \right] \right\} (I_{i+1})$$

where

$\delta \rightarrow$ deflection

$n \rightarrow$ number of sections from top in which

the column is suitable divided.

$W \rightarrow$ wind load on the section

$\Sigma W \rightarrow$ concentrated load on the section
 (due to moment of the section
 above)

$I \rightarrow$ moment of inertia of the cross
 section

$E \rightarrow$ modulus of elasticity

The liquid phase is dispersed in the gas phase. The droplets are small and have a high surface area. The droplets are dispersed in the gas phase. The droplets are small and have a high surface area. The droplets are dispersed in the gas phase. The droplets are small and have a high surface area.

COLUMN INTERNAL DETAILS

Various components such as internals, located inside a distillation column are designed to create an intimate contact between the vapour and liquid streams flowing evenly in opposite directions. According to the mode of contact between vapour and liquid, columns can be classified as equilibrium stage columns and differential column. The principal mass transfer between phases takes place in tray or a series of plates or trays at intervals of about 1m. In differential column, mass transfer between phases takes place gradually throughout the entire height of column.

Equilibrium stage column
Distillation column
Distillation column

The column tray spacing is determined by the column size. Columns of 300 mm and above spacings of 450 mm are preferred for ease of tray assembly when the vessel column diameter is less than 750 mm. Greenhouse tray installation becomes impossible and a central assembly is necessary. These are expensive and difficult to maintain, in particular if corrosion of fouling is expected. For 600 mm diameter, a differential column (packed column) is preferable.

Equilibrium stage column

Depending on the method of the method of contact and the flow of vapour and liquid, the contacting equipment may be classified as follows:

Plates without downcomers

These are generally flat circular plates of a diameter slightly smaller than the internal diameter of column. They are provided with a large number of holes or slots through which the liquid and vapour pass.

The tubed tray is a flat plate with long rectangular slots. The grid can also be formed by closely spaced parallel bars located in one cylindrical plane, instead of single flat plate.

PLATES WITH DOWNCOMERS These are circular flat plates or trays with provision of passages known as downcomers for the downward flow of the liquid. The downcomer area for each plate will be generally limited to 10% of the total area of the plate.

a) Downcomer types Internal External
The function of the downcomer is to provide a passage for the downward flow of the liquid from a top tray to a lower tray. A projection of a corresponding shape, provided on the downcomer acts as an exit weir. A radial flow is created by downcomers placed at the centre and around the circumference of alternate trays. Multiple downcomer trays are also designed.

b) Contacting devices

i) Bubble caps: → The area of the plate or tray between the inlet and exit

Weld is provided with a large number of holes over which ballast, mesh, or other materials are placed. Caps in the form of inverted cones located over the holes.

(ii) valves

These permit efficient mixing of liquid and vapor over a wide range loading.

(iii) Sieve plates

There is no separate contacting device in these types of plates. Instead of large number of holes distributed uniformly over the plate act as passages for vapours but at the same time the liquid is prevented from draining through the holes.

(c) Distribution baffles.

The spacing by 2.5 cm, the gaps are closed by redistribution baffles. The clearance between these baffles and the caps should be equal to the cap spacing.

FEED SYSTEMS

Feed nozzles are sized for a flow velocity of 1 meter per second. Liquid feed to the column is above the top tray and at intermediate position. A pipe is located above the top tray, and just opposite the inlet neck, feeds to top tray. A baffle is provided opposite the feed pipe to create

DIFFERENTIAL COLUMN

Packed Column: - Instead of using plates of trays, this column is Packed with suitable packing material. The liquid flows downward over the surface of the packing in the form of thin films, while the vapour rises, the liquid is introduced at the top of packing by means of a distributor tray plate and the vapour is introduced beneath a grid which supports the packing. Packed columns are particularly useful in the field of vacuum distillation. In corrosion problems packing are most suitable.

LIQUID DISTRIBUTORS :-

Various methods are adopted for distribution of the liquid. To effect proper wetting of the packing, pipe arrangement, flat perforated plates, or wiper type distributor are commonly used. For column diameters less than 60 cm a single pipe nozzle,

LIQUID REDISTRIBUTION AND WALL WIPERS

Despite its efficient distribution on the top plate, the liquid has tendency to drift towards the wall as it moves downwards. After a certain vertical height of liquid travel, liquid concentration falls considerably in the inner core section.

DIFFERENTIAL COLUMN
It is important to note that the liquid after the distributor travels a certain distance. Redistributors and wall wipers are provided to bring the liquid back to the centre from the wall.

SUPPORT PLATES

These plates are located at the bottom of the column and support the packings. Conventional type of support plate is a perforated plate.

HOLD-DOWN PLATES

A hold-down plate on top of a packed section serves an important purpose by resting directly on the free area of the plate is decided between as large a value of the area as possible and a value small enough to prevent the packing passing through the plate.